MIZORAM PUBLIC SERVICE COMMISSION

COMPETITIVE EXAMINATIONS FOR RECRUITMENT TO THE POST OF Inspector of Factories under Labour, Employment, Skill Development & Entrepreneurship Department, Government of Mizoram, 2019

MECHANICAL ENGINEERING PAPER - I

Time Allowed: 2 hours Full Marks: 200 All questions carry equal marks of 2 each. Attempt all questions. 1. A closed system is one in which -(a) both energy and mass cross the boundary of the system (b) the mass does not cross the boundary, but energy interaction takes place (c) neither mass nor energy cross the boundary of the system (d) the mass crosses the boundary but energy does not 2. When two bodies are in thermal equilibrium with a third body, then they are in thermal equilibrium with each other. This statement is called -(a) first law of thermodynamics (b) second law of thermodynamics (c) third law of thermodynamics (d) zeroth law of thermodynamics 3. An adiabatic system is one in which -(a) both energy and mass cross the boundary of the system (b) the mass does not cross the boundary, but energy interaction takes place (c) neither mass nor energy cross the boundary of the system (d) mass crosses the boundary, heat energy does not cross the boundary of the system **4.** Which of the following is an extensive property? (a) Volume (b) Temperature (c) Pressure (d) Density 5. The first law of thermodynamics deals with -(a) heat and work (b) quality of energy (d) measurement of energy transfer (c) balance of quantity of energy 6. The perpetual motion machine of the first kind is impossible according to the -(a) zeroth law of thermodynamics (b) first law of thermodynamics (c) second law of thermodynamic (d) third law of thermodynamics 7. The specific volume of water during freezing -(b) remains constant (a) Increases (d) none of these (c) decreases

(b) volume remains constant

(d) enthalpy change is maximum

8. In an isothermal process -

(a) temperature increases gradually

(c) change in internal energy is zero

| 9. | . In a reversible adiabatic process, the work transfer is equal to - | | | | |
|-----|---|---|-------|--|--|
| | (a) decrease in enthalpy | (b) decrease in internal energy | | | |
| | (c) heat transfer | (d) the product of pressure and change in vo | lume | | |
| 10. | . A control volume refers to - | | | | |
| | (a) a fixed region in space | (b) a fixed quantity of matter | | | |
| | (c) an isolated system | (d) a closed system | | | |
| 11. | . Steady flow occurs when - | | | | |
| | (a) properties do not change with time | (b) the system is in equilibrium with its surroun | dings | | |
| | (c) properties change with time | (d) when $\left(\frac{\partial v}{\partial t}\right)$ is constant | | | |
| 12. | . During a throttling process - | | | | |
| | (a) internal energy remains constant | (b) enthalpy of fluid remains constant | | | |
| | (c) pressure remains constant | (d) temperature remains constant | | | |
| 13. | . The second law of thermodynamics deals w | th - | | | |
| | (a) direction of process and quality of end | | | | |
| | (c) balance of internal energy | (d) system efficiency | | | |
| 14. | . It is impossible to construct an engine whi | h while operating in a cycle, produces no other e | ffect | | |
| | <u> </u> | perature reservoir and do equivalent amount of wo | | | |
| | (a) It refers to Clasius statement | (b) It refers to Kelvine-Planck's statement | nt | | |
| | (c) It refers to Carnot's theorem | (d) It refers to Clasius's theorem | | | |
| 15. | . A Carnot cycle comprises of - | | | | |
| | (a) two isothermal and two isentropic pro | cesses | | | |
| | (b) two constant volume and two isentrop | ic processes | | | |
| | (c) two constant pressure and two isentre | pic processes | | | |
| | (d) one constant volume, one constant pr | essure and two isentropic processes | | | |
| 16. | 6. In a thermal power plant, turbine does 10,000 kJ of work, pump consumes 10 kJ of work. To boiler receives 30,000 kJ of heat. Thermal efficiency of the plant is - | | | | |
| | (a) 27% | (b) 33.3% | | | |
| | (c) 35% | (d) 40% | | | |
| 17. | 7. A heat engine receives heat from a source at 1000°C and rejects the waste heat to a sink at 50°C. the heat is supplied to the engine at the rate of 100 kW. The maximum power output of this engine is | | | | |
| | (a) 25.48 | (b) 55.44 | | | |
| | (c) 74.62 | (d) 79.85 | | | |
| 18. | . Entropy is - | | | | |
| | (a) an extensive property | (b) an abstract property | | | |
| | (c) a function of quality of heat | (d) all of these | | | |
| 19. | Entropy of water at 0°C is assumed to be - | | | | |
| | (a) 0 | (b) 1 | | | |
| | (c) -1 | (d) none of these | | | |
| 20. | Entropy is a function of - | | | | |
| | (a) work transfer | (b) volume | | | |
| | (c) temperature | (d) pressure | | | |

| 21. | The av | ailable energy is - | | | | |
|-----|---------|---|--------|---------------------------------------|--|--|
| | (a) h | nigh-grade energy | (b) | portion of energy as useful work | | |
| | (c) tl | heoretical maximum amount of work | (d) | none of these | | |
| 22. | The de | gradation of energy is responsible for - | | | | |
| | (a) e | entropy generation within the system | (b) | decrease of entropy within the system | | |
| | (c) n | naximum work done by the system | (d) | none of these | | |
| 23. | Maxwe | ell's Thermodynamic relations are valid for - | | | | |
| | (a) a | ll processes | (b) | a closed system | | |
| | (c) a | thermodynamic system in equilibrium | (d) | an open system | | |
| 24. | A relat | ion of vapour pressure to enthalpy of vaporis | satio | n is expressed in - | | |
| | (a) v | an der waals equation | (b) | Maxwell's relations | | |
| | (c) (| Carrier's equation | (d) | Clausius-Claypeyron equation | | |
| 25. | Which | statement is true for Otto cycle? | | | | |
| | (a) H | Heat addition at constant volume and heat rej | ectio | n at constant volume | | |
| | (b) I | Heat addition at constant volume and heat rej | jectio | on at constant pressure | | |
| | (c) F | Heat addition at constant pressure and heat re | ejecti | on at constant volume | | |
| | (d) F | Heat addition at constant pressure and heat re | eject | ion at constant pressure | | |
| 26. | Air sta | ndard Diesel cycle consists of - | | | | |
| | (a) t | a) two isothermal and two constant-volume processes | | | | |
| | (b) t | wo isentropic and two constant-pressure pro | ocess | es | | |
| | (c) t | wo isentropic and two constant-volume proc | esse | S | | |
| | (d) n | none of these | | | | |
| 27. | The eff | ficiency of air standard Otto cycle depends of | n - | | | |
| | (a) p | pressure ratio in the cycle | (b) | temperature ratio in the cycle | | |
| | (c) (| Compression ratio in the cycle | (d) | mean effective pressure | | |
| 28. | Which | one of the following is part of air standard A | tkins | on cycle? | | |
| | ` ' | sothermal heat addition | (b) | J | | |
| | (c) (| Constant-volume heat addition | (d) | Constant-pressure heat addition | | |
| 29. | Which | one of the following is part of air standard B | rayto | on cycle? | | |
| | (a) F | Polytropic compression | (b) | Isochoric heat addition | | |
| | (c) I | sobaric heat addition | (d) | Isochoric and isobaric heat addition | | |
| 30. | The ma | ain constituents of a fuel are - | | | | |
| | | ydrogen and oxygen | (b) | • • | | |
| | (c) s | ulphur and hydrogen | (d) | sulphur and oxygen | | |
| 31. | Stoichi | ometric air-fuel ratio of petrol is roughly - | | | | |
| | (a) 5 | 50:1 | (b) | 25:1 | | |
| | (c) 1 | 5:1 | (d) | 1:1 | | |
| 32. | In a tw | o-stroke engine, one power stroke is obtain | ed in | ı - | | |
| | (a) o | one revolution of the crank shaft | (b) | two revolutions of the crank shaft | | |
| | (c) f | our revolutions of the crank shaft | (d) | none of these | | |

| 33. | In a I | In a Diesel engine, fuel consumption against brake power is - | | | | |
|-----|--|--|-------|--|--|--|
| | (a) | parabolic | (b) | linear | | |
| | (c) | hyperbolic | (d) | non-predictable | | |
| 34. | By us | y use of cooling, which efficiency of an IC engine decreases - | | | | |
| | (a) | volumetric efficiency | (b) | mechanical efficiency | | |
| | (c) | charging efficiency | (d) | thermal efficiency | | |
| 35. | Whic | ch one of following is a governing method used | on p | etrol engines? | | |
| | (a) | Quality governing | (b) | Quantity governing | | |
| | (c) | Injection governing | (d) | Hit and miss governing | | |
| 36. | Incre | ase in compression ratio in an Otto cycle engi- | ne m | ay cause - | | |
| | (a) | misfiring | (b) | detonation | | |
| | (c) | knocking | (d) | longer ignition delay | | |
| 37. | Inag | gas turbine plant, the intercooler is used in bet | ween | - | | |
| | (a) | air compressor and regenerator | (b) | air compressor and combustion chamber | | |
| | (c) | combustion chamber and turbine | (d) | LP compressor and HP compressor | | |
| 38. | The f | function of regenerator in a gas turbine plant is | _ | | | |
| | | to heat the compressed air from the compress | | | | |
| | | (b) to heat the gas before inlet to gas turbine | | | | |
| | (c) | (c) to exchange the heat from hot gases from combustion chamber to exhaust gases of the turbin | | | | |
| | (d) | to heat the compressed air in between the sta | ges | | | |
| 39. | Then | mal efficiency of an IC engine indicates percen | ıtage | of- | | |
| | (a) | BP converted into IP | (b) | heat converted into work | | |
| | (c) | IP converted into BP | (d) | heat lost into exhaust | | |
| 40. | A rocket engine receives oxygen for combustion of fuel from - | | | | | |
| | | oxidizer on board | | compressed atmospheric air | | |
| | (c) | surrounding air | (d) | none of the above | | |
| 41. | The absorptivity of thermal radiation by a solid surface can be enhanced - | | | | | |
| | (a) | by polishing the surface | (b) | by roughening the surface | | |
| | (c) | by increasing the surface area | (d) | by decreasing the surface area | | |
| 42. | Trans | sient heat conduction means - | | | | |
| | (a) conduction when the temperature at a point varies with time | | | | | |
| | (b) | heat conduction for a short time | | | | |
| | (c) | very little heat transfer | | | | |
| | (d) heat conduction with a very small temperature difference | | | | | |
| 43. | For a | balanced counter flow heat exchanger, the to | empe | erature profiles of the two fluids along the | | |
| | length of the heat exchanger - | | | | | |
| | (a) | linear | (b) | parallel | | |
| | (c) | linear and parallel | (d) | parabolic | | |
| 44. | A hea | at pipe is used to transfer heat from the source | to th | e sink by a fluid by means of | | |
| | (a) | conduction | (b) | evaporation | | |
| | (c) | condensation | (d) | evaporation and condensation | | |

| 45. Gases have poor - | | | | |
|---|--|--|--|--|
| (a) tranmissitivity | (b) absorptivity | | | |
| (c) reflectivity | (d) emissivity | | | |
| 46. COP of a Carnot refrigeration cycle is greater that | an - | | | |
| (a) vapour compression cycle | (b) reversed Brayton cycle | | | |
| (c) vapour absorption cycle | (d) all the above | | | |
| 47. In an ideal vapour compression refrigeration cycle | , the refrigerant is in the form of superheated vapour | | | |
| before entering into - | | | | |
| (a) condenser | (b) throttle valve | | | |
| (c) evaporator | (d) compressor | | | |
| 48. Heat is absorbed by a refrigerant during a refrige | erant cycle - | | | |
| (a) condenser | (b) throttle valve | | | |
| (c) evaporator | (d) compressor | | | |
| 49. A zeotropes are mixture of - | | | | |
| (a) primary and secondary refrigerant | (b) Ammonia and water | | | |
| (c) CFCs and HFCs | (d) HCFCs and HFCs | | | |
| 50. Subcooling of refrigerant in vapour compression | refrigeration cycle - | | | |
| (a) decreases COP | (b) increases COP | | | |
| (c) decrease refrigerating effect | (d) increases work input | | | |
| 51. In a forced vortex | | | | |
| (a) the fluid velocity is inversely proportional to the radius | | | | |
| (b) the fluid rotates without any relative velocit | У | | | |
| (c) the rise depends on the specific weight | | | | |
| (d) the rise is proportional to the cube of angul | ar velocity | | | |
| 52. The centre of pressure of a rectangular plane with height of liquid h m from base - | | | | |
| (a) $h/2$ m from bottom | | | | |
| (b) $h/3$ m from top | | | | |
| (c) h/3 m from bottom | | | | |
| (d) can be determined only if liquid specific we | eight is known | | | |
| 53. The location of the centre of pressure over a surf | ace immersed in a liquid is - | | | |
| (a) always above the centroid | | | | |
| (b) will be at the centroid | | | | |
| (c) will be below the centroid | | | | |
| (d) for higher densities it will be above the ce centroid | entroid and for lower densities it will be below the | | | |
| 54. If a body is in stable equilibrium the metacentric | height should be - | | | |
| (a) zero | (b) positive | | | |
| (c) negative | (d) depends on the fluid | | | |
| 55. When a ship leaves a river and enters the sea - | | | | |
| (a) It will rise a little | (b) It will sink a little | | | |
| (c) There will be no change in the draft | (d) It will depend on the type of the ship | | | |

| 56. | The | continuity equation is the result of application of | of the | following law to the flow field - |
|------------|--------|--|------------|--|
| | | First law of thermodynamics | | Conservation of energy |
| | (c) | Newton's second law of motion | (d) | Conservation of mass |
| 57. | Thes | stream function is - | | |
| | (a) | constant along an equipotential line | (b) | along a stream line |
| | (c) | defined only in irrotational flow | (d) | defined only for incompressible flow |
| 58. | Bern | oulli equation is applicable for - | | |
| | (a) | steady rotational flow | (b) | steady rotational compressible flow |
| | (c) | steady irrotational incompressible flow | (d) | unsteady irrotational incompressible flow |
| 59. | In a f | flow along a varying flow cross section, as the | e area | decreases - |
| | (a) | the energy line will slope up | (b) | the hydraulic gradient line will slope up |
| | (c) | the hydraulic gradient line will slope down | (d) | the energy line will slope down |
| 60. | In ste | eady flow in a varying section pipe if the diame | eter is | doubled the kinetic energy will - |
| | (a) | be doubled | (b) | increase 4 times |
| | (c) | increase 8 times | (d) | decrease to one sixteenth |
| 61. | Reyn | olds number signifies the ratio of - | | |
| | (a) | gravity forces top viscous forces | (b) | inertial forces to viscous forces |
| | (c) | inertia forces to gravity forces | (d) | buoyant forces to inertia forces |
| 62. | The | entry length in pipe flow will be higher for - | | |
| | (a) | highly viscous fluids | (b) | low viscosity fluid |
| | (c) | high velocity of flow | (d) | small diameters |
| 63. | | lly developed turbulent flow, if the diameter is hawill change by the factor - | alved | without changing the flow rate, the frictional |
| | (a) | 32 times | (b) | 16 times |
| | (c) | 8 times | (d) | 4 times |
| 64. | The v | velocity profile in turbulent flow is - | | |
| | (a) | parabolic | (b) | logarithmic |
| | (c) | 2 nd degree polynomial | (d) | 4th degree polynomial |
| 65. | A pit | tot static tube is used to measure - | | |
| | (a) | Stagnation pressure | | |
| | (b) | Static pressure | | |
| | | Dynamic pressure | | • |
| | ` ' | Difference between the static pressure and dy | /nam | ic pressure |
| 00. | | meter is used to measure - | (h) | Elaw |
| | | Viscosity Density | ` / | Flow Pressure |
| | ` , | • | (u) | riessure |
| 67. | | ficient of discharge is the ratio of - | (1-) | The anatical flarm/A street flarm |
| | ` ' | Actual flow/Theoretical flow | (b) | Theoretical flow/Actual flow Theoretical value its: Theoretical value its: |
| (0 | ` ' | Actual velocity/Theoretical velocity | (d) | Theoretical velocity/Actual velocity |
| 08. | - | in the case of a centrifugal pump. | (L) | Padvage the energy transfer |
| | ` / | Reduces the flow rate Reduces the speed | (b) (d) | Reduces the energy transfer Increases cavitation |
| | (0) | reduces the speed | (4) | moreases environ |

| 69. | A lov | w specific speed Francis turbine is - | | | |
|------------|---|---|--------|---------------------------------------|--|
| 0,7 | | Axial flow turbine | (b) | tangential flow turbine | |
| | ` / | Mixed flow turbine | ` ′ | radial flow turbine | |
| 70. | ` / | nple of a pure reaction turbine is - | () | | |
| . •• | | Francis turbine | (b) | Propeller turbine | |
| | ` / | Kaplan turbine | ` ′ | Lawn sprinkler | |
| 71. | ` ' | on turbine is a - | () | • | |
| , 1, | | Reaction turbine | (b) | Impulse turbine | |
| | ` / | Radial flow turbine | ` ′ | Axial flow turbine | |
| 72. | ` / | normal shock taking place in a gas - | () | | |
| | | the velocity, pressure and density increase ac | cross | the shock | |
| | | the entropy remains constant | | | |
| | ` ' | the entropy decreases across the shock | | | |
| | | the entropy increases across the shock | | | |
| 73. | The f | function of superheated is to - | | | |
| | (a) | Superheat the steam | (b) | Extinguish the fire | |
| | (c) | Maintain constant temperature | (d) | Preheat the feed water | |
| 74. | Whic | ch one of the following is the correct sequence | of ac | ecessories in a boiler plant? | |
| | (a) | Boiler-economiser-superheater-chimney | (b) | Economiser-boiler-superheater-chimney | |
| | (c) | Economiser-air preheater-superheater-chimn | ey(d) | Economiser-boiler-preheater-chimney | |
| 75. | The | draught in a boiler is provided to - | | | |
| | (a) | force the air on the furnace | (b) | force the hot gases on superheater | |
| | (c) | discharge the fiue gases through chimney | (d) | all of these | |
| 76. | The | equivalent evaporation is defined a - | | | |
| | (a) | steam generated at 100°C | | | |
| | (b) dry and saturated steam generated at 100°C from feed water at 100°C | | | | |
| | (c) steam generated at 1 bar and at 100°C | | | | |
| | (d) | none of these | | | |
| 77. | A fiu | id is a compressible fluid when its density - | | | |
| | | decreases with pressure | ` ' | increases with pressure | |
| | (c) | increases with temperature | (d) | both (a) & (b) | |
| 78. | | zzle is designed for - | | | |
| | ` ' | maximum pressure at outlet | | | |
| | | minimum pressure at outlet | | | |
| | | maximum discharge at outlet | | | |
| | . , | maximum discharge and maximum pressure a | t outl | et | |
| 79. | | zle efficiency is defined as the ratio of - | | | |
| | | actual enthalpy drop to isentropic enthalpy dr | - | | |
| | (b) isentropic enthalpy drop to actual enthalpy drop | | | | |
| | (c) product of isentropic enthalpy drop and actual enthalpy drop | | | | |

(d) square root of isentropic enthalpy drop to actual enthalpy drop

| 80. | When | n a fiuid is coming out of a duct at a higher pre | essure | e than it enters, the duct is called a/an - | | | | | |
|------------|------------------------------------|---|--------|---|--|--|--|--|--|
| | (a) | orifice | (b) | nozzle | | | | | |
| | (c) | diffuser | (d) | venturi | | | | | |
| 81. | Prese | ence of frictional effect during fiow through the | nozz | zle - | | | | | |
| | (a) | reduces the exit velocity | (b) | increases the exit velocity | | | | | |
| | (c) | has no effect on exit velocity | (d) | none of these | | | | | |
| 82. | The s | hear stress in turbulent flow is: | | | | | | | |
| | (a) | Linearly proportional to the velocity gradient | | | | | | | |
| | (b) | Proportional to the square of the velocity gradient | | | | | | | |
| | (c) | Dependent on the mean velocity of flow | | | | | | | |
| | (d) | Due to the exchange of energy between the m | olec | ules | | | | | |
| 83. | 0.071 the p | Water at 25°C is flowing through a 1.0 km long G.I. pipe of 200 mm diameter at the rate of $0.07 \text{m}^3/\text{s}$. If value of Darcy friction factor for this pipe is 0.02 and density of water is 1000 kg/m^3 , the pumping power (in kW) required to maintain the flow is: | | | | | | | |
| | ` / | 1.8 | () | 17.4 | | | | | |
| | (c) | 20.5 | (d) | 41.0 | | | | | |
| 84. | A pip | beline is said to be equivalent to another, if in b | oth - | | | | | | |
| | ` ' | Length and discharge are the same | | | | | | | |
| | | Velocity and discharge are the same | | | | | | | |
| | ` ' | Discharge and frictional head loss are the san | ne | | | | | | |
| | (d) | d) Length and diameter are the same | | | | | | | |
| 85. | Subs | onic and supersonic diffusers have the following | ng ge | ometry - | | | | | |
| | (a) | Divergent and convergent respectively | (b) | Both divergent | | | | | |
| | (c) | Both convergent | (d) | Convergent and divergent respectively | | | | | |
| 86. | Cavit | tation in a hydraulic turbine is most likely to oc | ecur a | at the turbine - | | | | | |
| | (a) | Entry | (b) | Exit | | | | | |
| | (c) | Stator exit | (d) | Rotor exit | | | | | |
| 87. | Eulei | equation for water turbine is derived on the b | asis (| of- | | | | | |
| | (a) | Conservation of mass | (b) | Rate of change of linear momentum | | | | | |
| | (c) | Rate of change of angular momentum | (d) | Rate of change of velocity | | | | | |
| 88. | Consider the following statements: | | | | | | | | |
| | 1. | 1. Pelton wheel is a tangential flow impulse turbine | | | | | | | |
| | 2. | 2. Francis turbine is an axial flow reaction turbine | | | | | | | |
| | 3. | 3. Kaplan turbine is a radial flow reaction turbine | | | | | | | |
| | Whic | Which of the above statements is/ are correct? | | | | | | | |
| | (a) | 1 and 3 | (b) | 1 alone | | | | | |
| | (c) | 2 alone | (d) | 3 alone | | | | | |
| 89. | The r | The ratio of work-done per cycle to the stroke volume of the compressor is known as - | | | | | | | |
| | (a) | Compressor capacity | (b) | Compression ratio | | | | | |
| | (c) | Compressor efficiency | (d) | Mean effective pressure | | | | | |

| 90. | A Pelton | wheel | is ideal | lly suited for |
|-----|----------|-------|----------|----------------|
|-----|----------|-------|----------|----------------|

- (a) High head and low discharge
- (b) High head and high discharge
- (c) Low head and low discharge
- (d) Medium head and medium discharge
- **91.** Thermal diffusivity of a substance is:
 - (a) Inversely proportional to thermal conductivity
 - (b) Directly proportional to thermal conductivity
 - (c) Directly proportional to the square of thermal conductivity
 - (d) Inversely proportional to the square of thermal conductivity
- **92.** In which one of the following materials, is the heat energy propagation minimum due to conduction heat transfer?
 - (a) Lead

(b) Copper

(c) Water

(d) Air

93. Assertion (A): The leakage heat transfer from the outside surface of a steel pipe carrying hot gases is reduced to a greater extent on providing refractory brick lining on the inside of the pipe as compared to that with brick lining on the outside.

Reason (R): The refractory brick lining on the inside of the pipe offers a higher thermal resistance.

- (a) Both A and R are individually true and R is the correct explanation of A
- (b) Both A and R are individually true but R is not the correct explanation of A
- (c) A is true but R is false
- (d) A is false but R is true
- **94.** Two insulating materials of thermal conductivity K and 2K are available for lagging a pipe carrying a hot fluid. If the radial thickness of each material is the same.
 - (a) Material with higher thermal conductivity should be used for the inner layer and one with lower thermal conductivity for the outer.
 - (b) Material with lower thermal conductivity should be used for the inner layer and one with higher thermal conductivity for the outer.
 - (c) It is immaterial in which sequence the insulating materials are used.
 - (d) It is not possible to judge unless numerical values of dimensions are given.
- 95. Upto the critical radius of insulation:
 - (a) Added insulation increases heat loss
 - (b) Added insulation decreases heat loss
 - (c) Convection heat loss is less than conduction heat loss
 - (d) Heat flux decreases
- **96.** Match List-I with List-II and select the correct answer using the codes given below the lists:

| <u>List-I</u> | <u>List-II</u> | |
|----------------------|--------------------------|--|
| A. Momentum transfer | 1. Thermal diffusivity | |
| B. Mass transfer | 2. Kinematic viscosity | |
| C. Heat transfer | 3. Diffusion coefficient | |
| (a) A-2, B-3, C-1 | (b) A-1, B-3, C-2 | |
| (c) A-3, B-2, C-1 | (d) A-1, B-2, C-3 | |

- **97.** A copper block and an air mass block having similar dimensions are subjected to symmetrical heat transfer from one face of each block. The other face of the block will be reaching to the same temperature at a rate:
 - (a) Faster in air block
 - (b) Faster in copper block
 - (c) Equal in air as well as copper block
 - (d) Cannot be predicted with the given information
- 98. A bomb calorimeter is used to determine -
 - (a) higher calorific value of solid or liquid fuel
- (b) lower calorific value of solid or liquid fuel
- (c) higher calorific value of gaseous fuel
- (d) lower calorific value of gaseous fuel
- 99. The gas having higher calorific value is -
 - (a) water gas

(b) coke-oven gas

(c) blast-furnace gas

(d) producer gas

- 100. A good fuel has -
 - (a) low ignition point and high calorific value
- (b) low ignition point and low calorific value
- (c) high ignition point and high calorific value
- (d) high ignition point and low calorific value

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